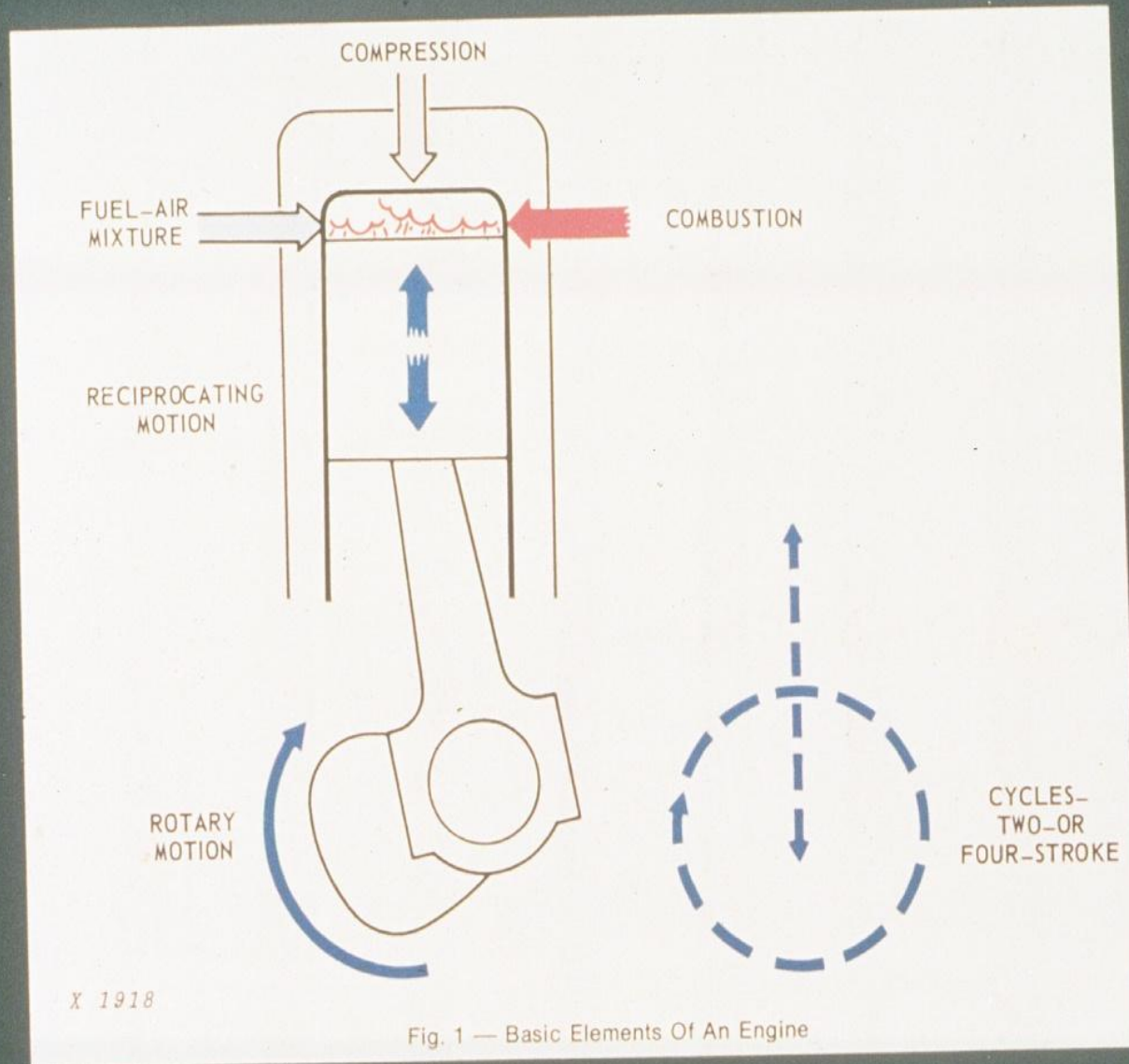


Engine Principles (Part 1)



Gas Laws

Ideal Gas Equation

$$pV = MRT$$

p = Pressure (kPa, psi), V = volume (m³, ft³), M = Mass (kg, lb), T = Absolute Temperature (K, R)

R = specific gas constant (8.314/molecular weight [metric], 10.72/molecular weight [English])

- Constant Volume

$$p_b/p_a = T_b/T_a$$

$$Q/M = C_v * (T_b - T_a)$$

- Constant Pressure

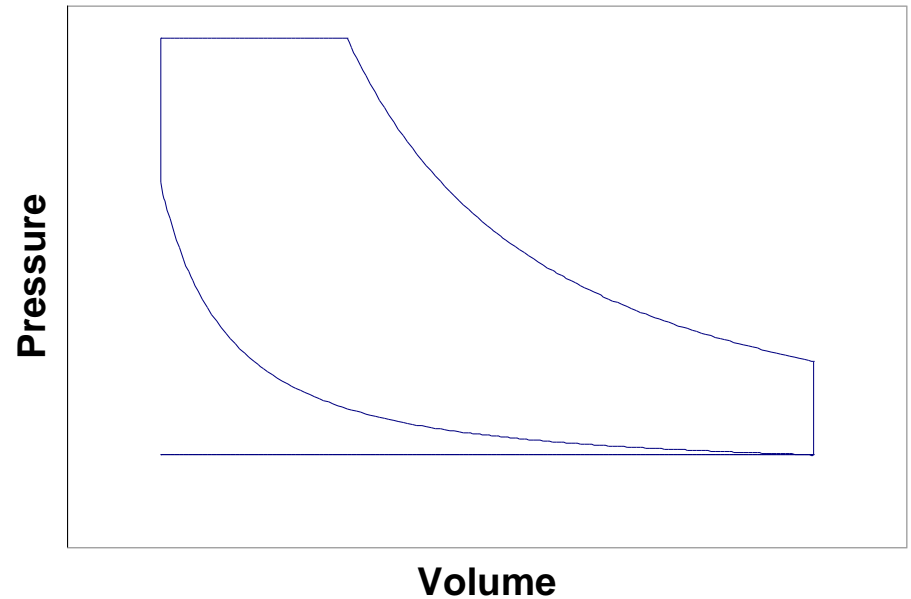
$$V_b/V_a = T_b/T_a$$

$$Q/M = C_p * (T_b - T_a)$$

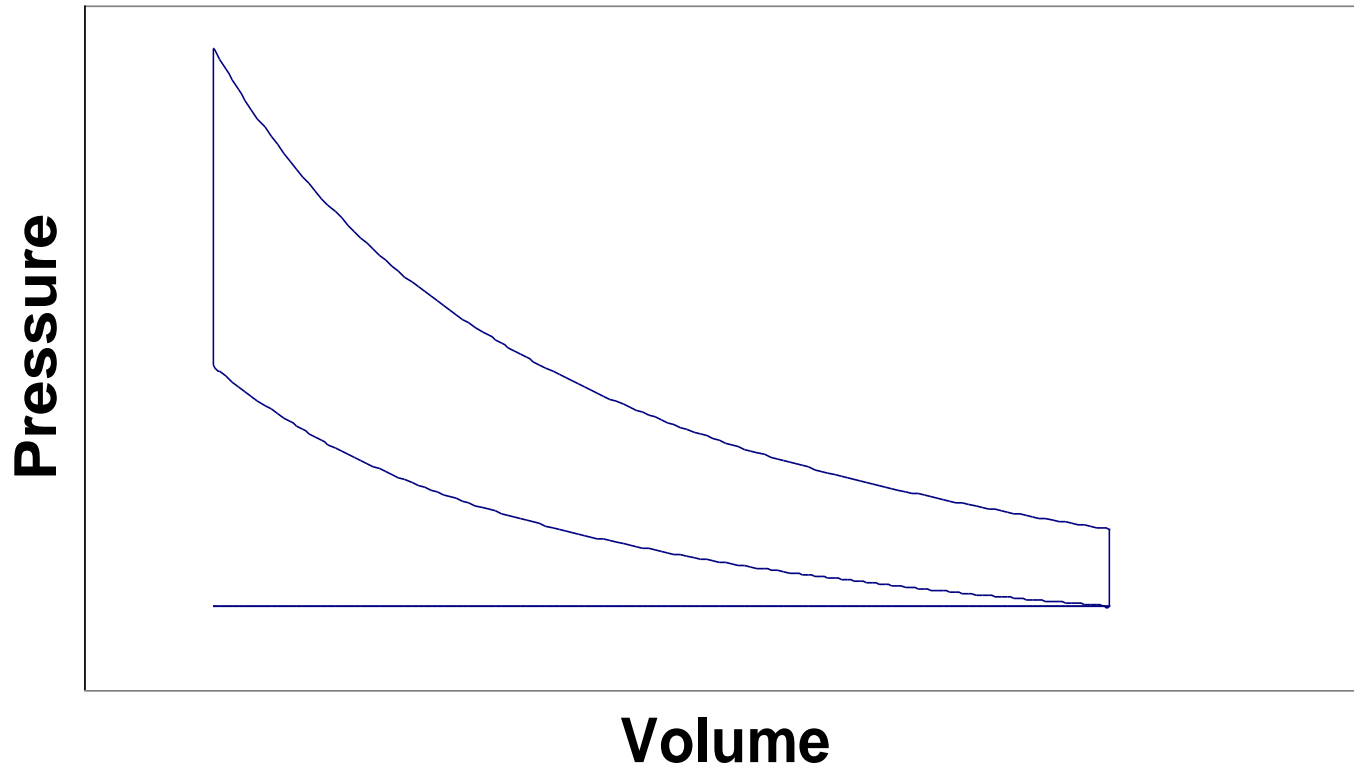
- Adiabatic Expansion/Compression

$$p_b/p_a = [V_a/V_b]^k = [T_b/T_a]^{k/(k-1)}$$

$$T_b/T_a = [V_a/V_b]^{(k-1)} = [p_b/p_a]^{(k-1)/k}$$

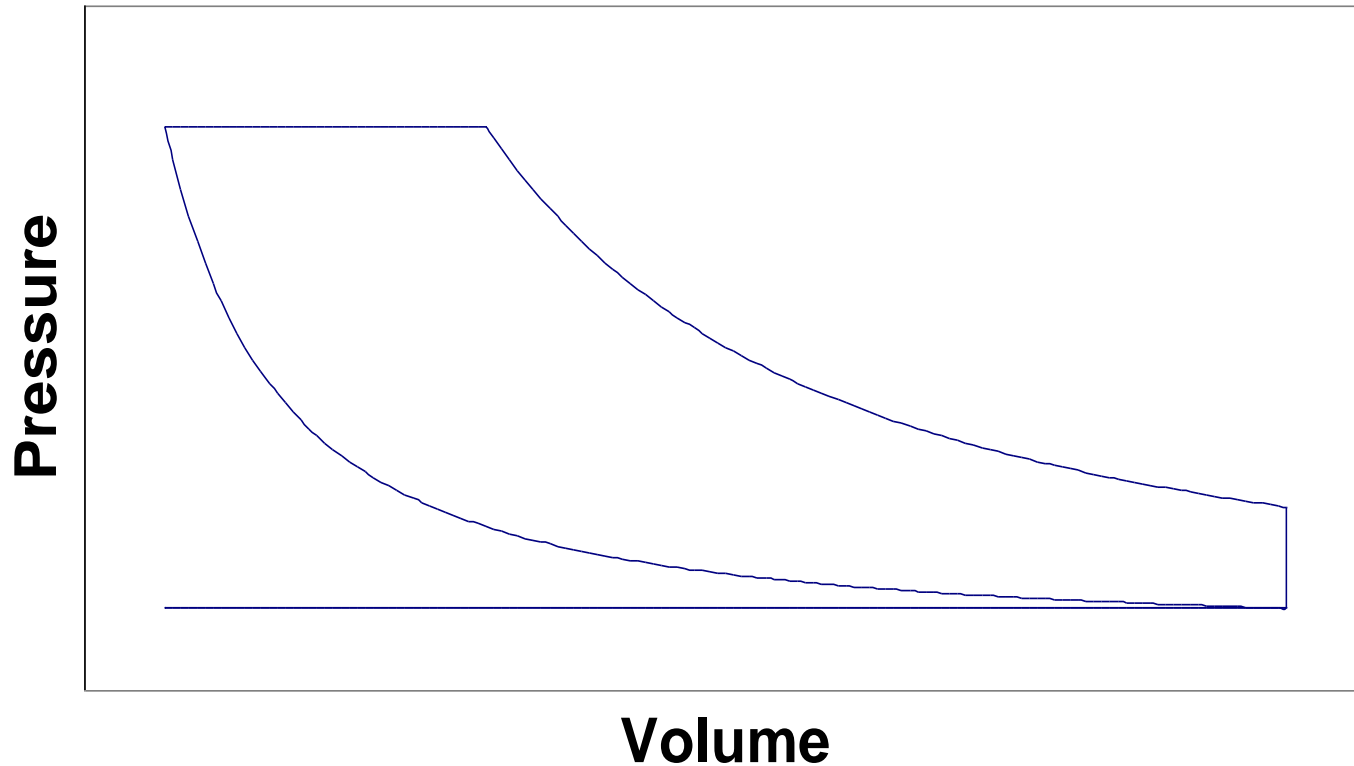


Theoretical Otto Cycle (SI)



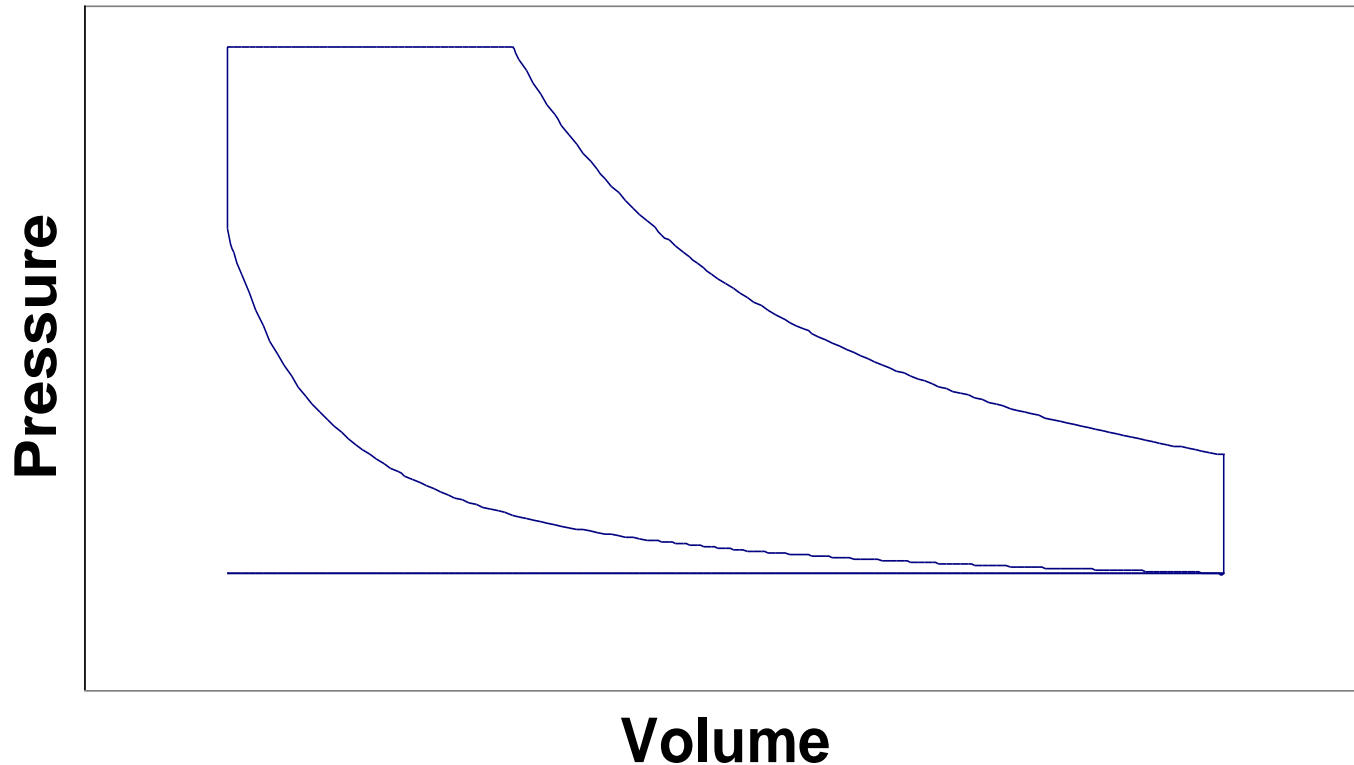
$$\text{Theoretical Efficiency} = 1 - \frac{1}{r^{(n-1)}}$$

Diesel Cycle (CI)



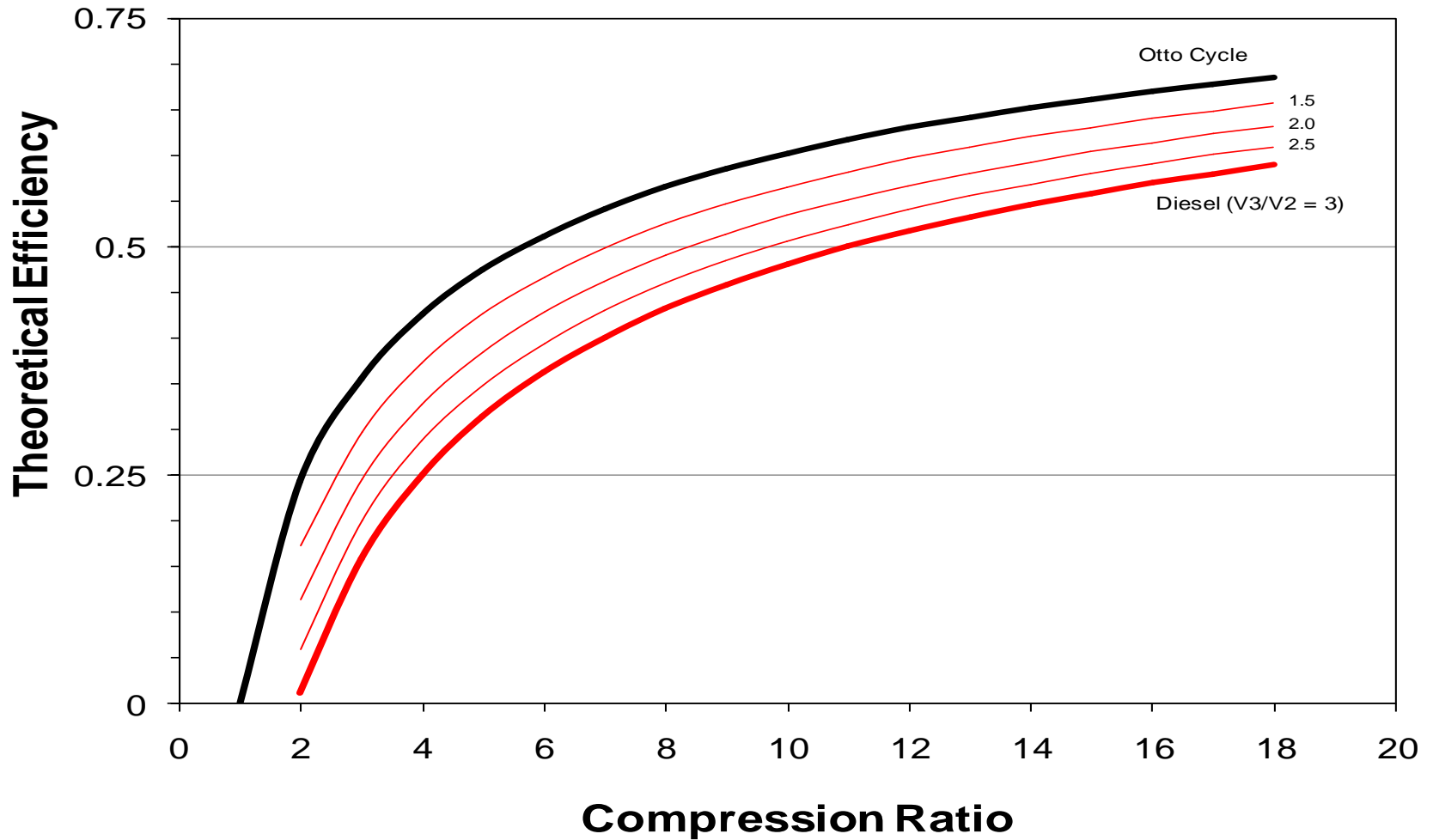
$$\text{Theoretical Efficiency} = 1 - \left[\frac{1}{r} \right]^{(n-1)} * \left[\frac{(r_{co})^n - 1}{n(r_{co} - 1)} \right]$$

Dual Cycle



$$\text{Efficiency} = 1 - \left[\frac{1}{r^{(n-1)}} \right] * \left[\frac{B(r_{co}^n - 1) + n(r_{co} - 1)(1 - B)}{n(r_{co} - 1)} \right]$$

Theoretical Cycle Efficiencies



Engine Cycles

- Otto Cycle

- Theoretical Efficiency = $1 - 1/r^{(n-1)}$

$$\underline{r = 8} \qquad \qquad \qquad \underline{\text{Eff} = 0.56}$$

- Diesel Cycle

- Theoretical Efficiency = $1 - [1/r^{(n-1)}] * [(r_{co})^n - 1] / [n(r_{co} - 1)]$

$$\underline{r = 8} \qquad r_{co} = 2.5 \qquad \underline{\text{Eff} = 0.46}$$

$$\underline{r = 16} \qquad r_{co} = 2.5 \qquad \underline{\text{Eff} = 0.59}$$

- Dual Cycle

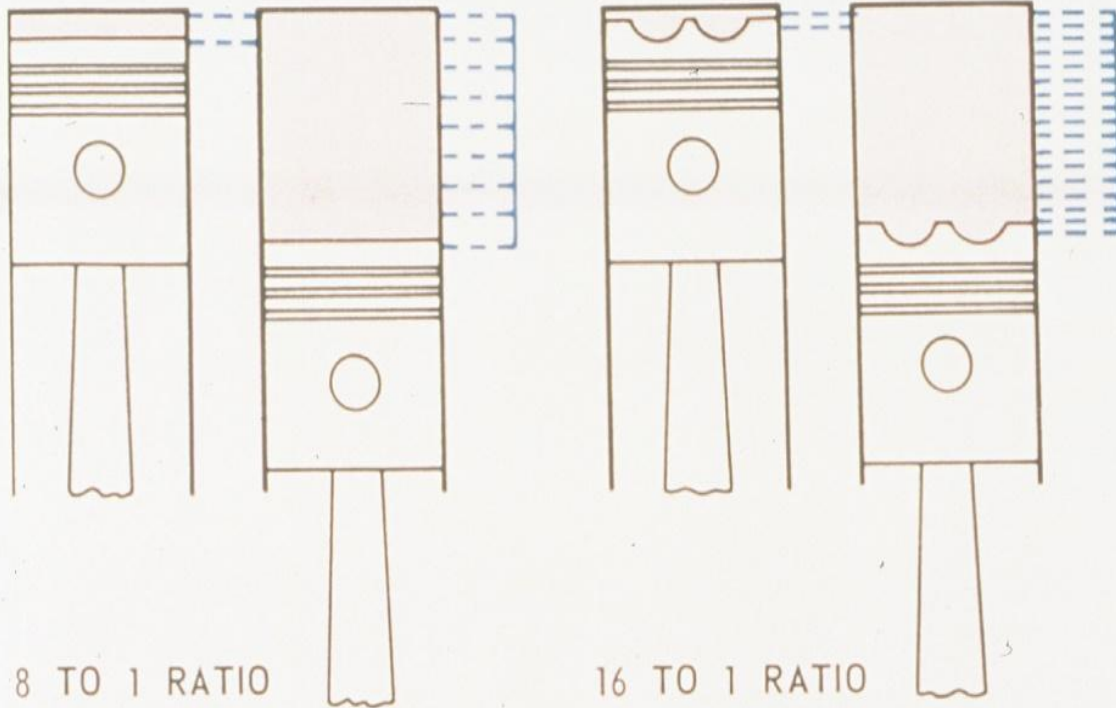
- Efficiency = $1 - [1/r^{(n-1)}] * \{ [B(r_{co})^n - 1] + n(r_{co} - 1)(1 - B) \} / [n(r_{co} - 1)]$

- Theoretical Efficiency

Lower than Otto Cycle for same compression ratio

Higher than Diesel Cycle for same compression ratio

COMPRESSION RATIOS: GASOLINE VS. DIESEL



8 TO 1 RATIO

16 TO 1 RATIO

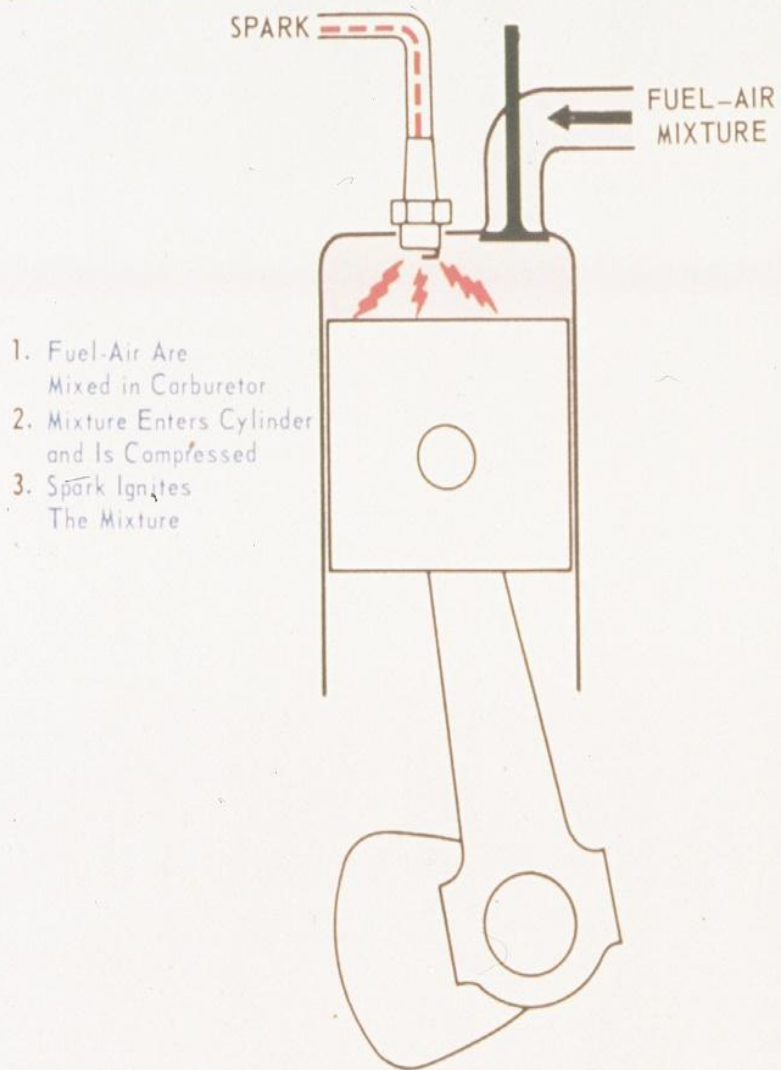
GASOLINE

DIESEL

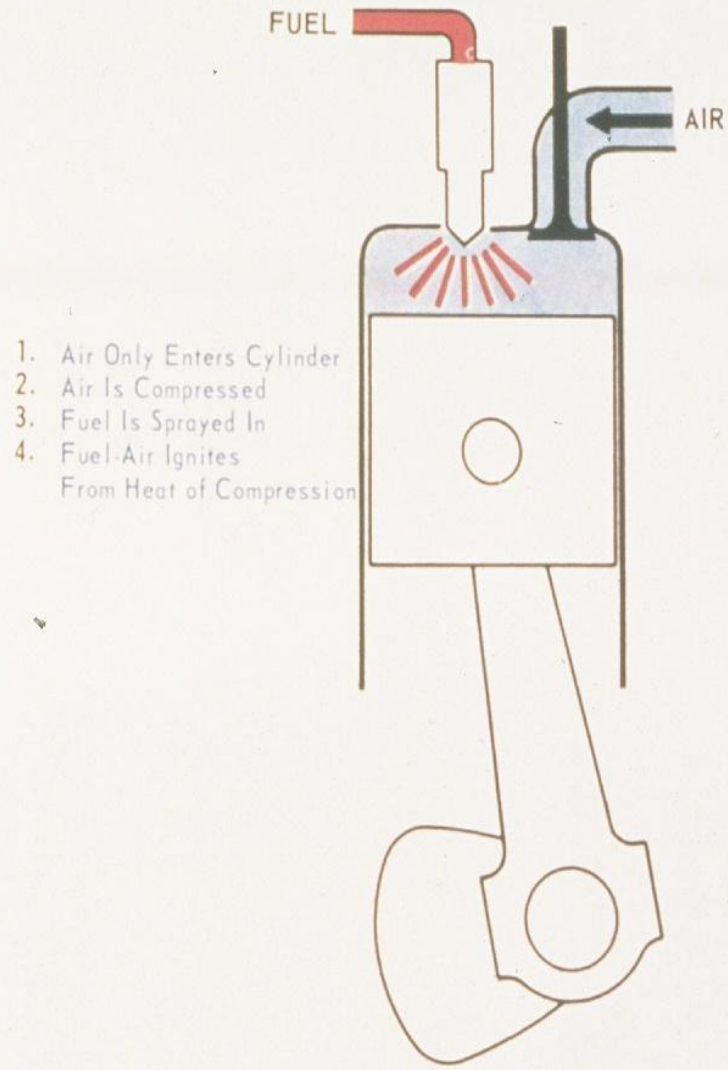
X 1935

Fig. 25 — Compression Ratios Compared

Differences Between Gasoline And Diesel Engines



GASOLINE



DIESEL

X 1934

Fig. 24 — Methods Of Supplying And Igniting Fuel

Characteristics of Four Stroke Compression Ignition & Spark Ignition Engines

<u>Characteristics</u>	<u>Compression-Ignition Engine</u>	<u>Spark- Ignition Engine</u>
Compression Ratio	14-22 : 1	5-8 : 1
Ignition	Compression	Electric Spark
Thermal Efficiency	30-60%	25-30%
Fuel induction	Injector	Carburettor (Fuel Injection)
Fuel System	Fuel Oil / Diesel	Gasoline (LP gas)
Fire Hazard	Less	Greater
Power Variation	Increase in Fuel	Increase in Air/Fuel Mixture
Air Induction	Constant	Variable (Throttle Airflow)
Air-Fuel Ratio	15-100 : 1	10-20 : 1
Relative Fuel Consumption	Lower	Higher
Energy per litre of fuel	Higher	Lower
Manifold Throttle	Absent	Present
Exhaust Gas Temperature	482° C / 900 F	704° C / 1300 F
Starting	Harder	Easier
Lubricants	Heavy duty oils	Regular and Premium Oils
Speed Range	Limited (600-3200 rpm)	Wide range (400-6000 rpm)
Engine Mass per Horsepower	8 kg (17.5 lb)	Average 4 kg (9 lb)
Initial Cost	High	Much Lower
Lugging ability (Torque)	Excellent	Less